

The operation of the brake rocker arm 300 during an engine braking operation will now be described. During engine braking, the solenoid valve 340 is operated. Hydraulic fluid is permitted to flow from passage 12 through passageway 360 to passageway 350. The actuator piston 320 then extends to a fully extended position such that it contacts pin 133 on crosshead 130. When the passageway 350 is filled with hydraulic fluid and the pressure is equalized within valve 340, a hydraulic lock is formed thus holding the actuator piston 320 in an extended position. The operation of the exhaust valve is now controlled by the brake rocker arm 300 in response to actuation by the brake cam lobe 23. The operation of the exhaust valves will occur in response to the profile of the brake cam lobe 23.

The brake cam lobe 23 also preferably has an exhaust gas recirculation lobe that occurs after the first braking event. This exhaust gas recirculation lobe on cam profile is disposed so that exhaust gas recirculation occurs after the first braking event, as shown in FIG. 3. Preferably, this allows the valves to remain open, which in turn allows exhaust gases to flow into the cylinder on the power stroke, charging the cylinder prior to the second braking event. The brake cam lobe 23 once again lifts the rocker arm just before exhaust top dead center, permitting a second braking event as shown in FIG. 3.

Effective two-cycle engine braking may be achieved in accordance with the present invention. The operating sequence of events will now be described. A first compression release cycle or braking event 1 is initiated just prior to compression top dead center, as shown in FIG. 3. The exhaust valve is then reset by partially closing the exhaust valve. The partial closing of the exhaust valve permits the recharging of the cylinder through an exhaust gas recirculation event 2, as shown in FIG. 3. The exhaust valve is then completely closed at the completion of the exhaust gas recirculation event. During this engine operating sequence, the normal operation of the exhaust valve by the exhaust rocker 100 is disabled. The operation of the at least one exhaust valve is controlled by the brake rocker arm 300. The profile of the brake cam lobe 23 initiates the first braking event 1 and causes the at least one exhaust valve to remain partially open during the exhaust gas recirculation event 2.

A second compression release cycle or braking event 3 is initiated just prior to exhaust top dead center, as shown in FIG. 3. The profile of the brake cam lobe 23 initiates the opening and closing of the at least one exhaust valve during the second braking event 3. The opening event 4 of the at least one intake valve is delayed past the exhaust top dead center, as shown in FIG. 3. The delayed intake valve opening prevents the valve to open when high cylinder pressure is present.

Alternate Embodiments

Continuing with the embodiments in the accompanying figures, FIG. 16 is an alternative embodiment for the means for effecting exhaust valve operation. The exhaust rocker arm 1000 is rotatably mounted on the common rocker shaft 11. A first end of the exhaust rocker arm 1000 includes an exhaust cam lobe follower 110.

A second end of the exhaust rocker arm 1000 has a lash adjuster 120. The lash adjuster 120 is connected adjacent to a crosshead 130. The crosshead 130 is preferably a bridge device that is capable of opening two valves simultaneously. The exhaust rocker arm 1000 also includes a solenoid valve 1400. The solenoid control valve 1400 is in communication with a fluid passageway 150 that extends through the

exhaust rocker arm 100 to the lash adjuster 120. The solenoid control valve 1400 is also in communication with a fluid passageway 160 that extends between the solenoid valve 140 and supply passage 13 of the common rocker shaft 11. The solenoid valve 1400 combines the valve 30 and the solenoid valve 140 into a single assembly.

FIG. 17 is an alternative embodiment for the means for effecting intake valve operation. The intake rocker arm 2000 is rotatably mounted on the common rocker shaft 11. A second end of the intake rocker arm 2000 has a lash adjuster 220. The intake rocker arm 2000 also includes a solenoid valve 2400. The solenoid valve 2400 is in communication with a fluid passageway 250 that extends through the exhaust rocker arm 2000 to the lash adjuster 220. The solenoid valve 2400 has the same construction as the solenoid valve 1400 described above in connection with the exhaust rocker arm 1000.

The intake rocker arm 2000 and the exhaust rocker arm 1000 operate in substantially the same manner as the intake rocker arm 200 and the exhaust rocker arm 100. In this embodiment, the solenoid valve 30 is eliminated.

It will be apparent to those skilled in the arts that various modifications and variations can be made in the construction and configuration of the present invention, without departing from the scope or spirit of the invention. Several variations have been discussed in the preceding text. Furthermore, it is contemplated that the present invention may be used with a common rail camless type engine whereby the above described rocker arms may be electronically operated. Others will be apparent to persons of ordinary skills in the art. It is intended that the present invention cover the modifications and variations of the invention, provided they come within the scope of the appended claims and their equivalents.

We claim:

1. An apparatus for performing multi-cycle engine braking, said apparatus comprising:

exhaust valve operating means for operating at least one exhaust valve of an engine cylinder during a positive power engine operation;

intake valve operating means for operating at least one intake valve of the engine cylinder; and

braking means for operating the at least one exhaust valve of the engine cylinder during an engine braking operation, wherein said braking means accomplishes at least two braking operations for the at least one exhaust valve per engine cycle during the engine braking operation, wherein said intake valve operating means delays the operation of the at least one intake valve during the engine braking operation.

2. The apparatus according to claim 1, wherein said means for operating the at least one exhaust valve during the positive power engine operation includes an exhaust rocker arm.

3. The apparatus according to claim 1, wherein said exhaust valve operating means includes exhaust valve engaging means for engaging the at least one exhaust valve to effectuate operation of the at least one exhaust valve.

4. The apparatus according to claim 3, wherein said exhaust valve engaging means releasably engages a crosshead pin of the at least one exhaust valve.

5. The apparatus according to claim 3, wherein said exhaust valve engaging means comprises a lash adjusting assembly.

6. The apparatus according to claim 5, wherein said lash adjusting assembly is hydraulically operated.

17. The apparatus according to claim 16, wherein said brake rocker arm disengages the crosshead pin during the positive power engine operation.

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